# VARIATIONAL FORMULATION IN THE GEOMETRICALLY NONLINEAR THIN ELASTIC SHELL THEORyt

## M. L. SZWABOWICZ

Institute of Fluid-Flow Machinery of the Polish Academy of Sciences, 80-952 Gdansk, Poland

#### *(Received* 22 *June* 1984; in *revisedform* 20 *September 1985)*

Abstract-The variational formulation for thin elastic shells undergoing large deflections is discussed. No restrictions are imposed on the magnitude of the rotational part of the deformation gradient, yet it is assumed that the strains remain small everywhere in the shell. Ten functionals and associated variational principles are derived, including the Hu-Washizu, the Hellinger-Reissner, the generalized complementary energy and the stationary potential energy principles. The total Lagrangian description (TLD) is exclusively used.

# I. INTRODUCfION

Within the theory of shells the variational approach was applied successfully first to its simplest linear version. The literature concerning this topic is large, and Alumyae, Aynola and Reissner, it seems, should be regarded as the earliest contributors to that field. For references, see the books by Washizu[1], Abovski et al.[2] and Rozin[3], which contains an exhaustive treatment of this and other related problems.

Since then, the variational setting for the shell theory was gradually extended to its more complex versions like the nonlinear theory of shallow shells (cf. the early paper by Aynola[4]) and, afterwards, the simplified versions of the moderate-rotation shell theory. Recent papers[5-8] may be mentioned here. The paper by Schmidt and Pietraszkiewicz[9] offers a profound study of variational principles within the complete version of the moderate-rotation shell theory. In [9] the Hu-Washizu, the Hellinger-Reissner and many other principles were derived on the basis of the principle of virtual work.

This paper will focus on the variational formulation of the general geometrically nonlinear first-approximation theory of thin elastic shells. This means that no restrictions are imposed on the rotational part of the deformation gradient (understood in the sense of the polar decomposition), but it is assumed that all the eigenvalues of the strain tensor are small in comparison with unity everywhere in the shell domain. Our chief goal is to pursue a self-consistent variational approach to such theory, based on the Kirchhoff-type of stress measures and the two-dimensional principle of virtual work as the fundamental static postulate. This task was partially undertaken by Galimov in [10, 11], where some principles of the Hu-Washizu, Reissner and Castigliano type were derived. However, those papers neglected the problem of the potentiality of the external loads applied to the shell-a question that appears, particularly in the case of the boundary moment, to be a factor limiting the possibilities of construction of the appropriate functionals. Besides, the functionaIs associated with those principles, though transformed to the undeformed configuration as a whole, still contained some variables related to the deformed shell boundary (cf. [12]) and thus were incompatible with purely Lagrahgian description. Variational principles equivalent to the ones presented in this paper were brought out by Schmidt in [13-16], where a whole family offunctionals was derived according to a procedure developed formerly in [9] for the moderate-rotation theory and the present author's earlier results[12, 17-20]. In particular, the functionals in [13-16] were based on a potential of the boundary forces and moments discussed in [17] and corresponding to the dead-load tractions at the lateral surface of the shell. This same type of potential was employed by the author in the

t This paper is based in part on a dissertation submitted by the author to the Institute of Fluid-Flow Machinery of the Polish Academy of Sciences in partial fulfillment of the requirements for the Ph.D. degree and on the material prescnted at the IIIrd Polish Conference-Shell Constructions-The Theory and Applications, Opole. 11-13 November. 1982.

functionals published in [12, 18-20]. Since this confines the applicability of the functionals in question only to this specific load, we discuss in detail the general condition for the existence of the boundary moment potential in this paper.

A different approach was suggested by Simmonds and Danielson[2l] and recently developed by Atluri[22] and Valid[23]. Their papers resort directly to the polar decomposition of the deformation gradient tensor and the functionals include pure stretches and rigid rotations together with the corresponding conjugate stress measures as the independent variable fields.

This paper extends the results presented in [12, 17-20], wherein the Hu-Washizu and subsequent variational principles were derived to a more general context. This work is particularly based on the papers [12,20] containing the relations ofthe general geometrically nonlinear theory of thin elastic shells reformulated in the entirely Lagrangian description with displacements as the main independent variables. However, in accordance with the remark made earlier we admit here a broader class of moments acting at the boundary of the shell. In the following section we repeat briefly the basic relations derived in  $[12, 20]$ . Then, assuming the principle of virtual work as the starting point, a procedure is followed to yield 10 functionals and associated variational principles. These functionals depend on combinations of strain, displacement and stress fields. At first an initial functional I, subject to some subsidiary conditions, is constructed. The resulting free functionals, eight in number, are grouped in two equivalent families  $\mathcal I$  and  $\mathcal J$ . The family  $\mathcal J$  contains the Hu-Washizu and the Hellinger-Reissner variational principles, whereas the family  $\ell$  includes the generalized complementary energy principle among the others. The accompanying free functionals form a basis suitable for deriving further functionals, with or without subsidiary conditions, and variational principles associated with them. For this end one should employ the well-known techniques concerning variational problems in general (cf.  $[1-3, 9, 24-26]$ ). To exemplify this we derive the functional of the total potential energy  $\Pi$ .

The reduction of the present results to the consistently simplified versions of the nonlinear thin elastic shell theory[27, 28] would lead to the appropriate families of variational principles and associated functionals, for example, it can be shown that if the rotations occurring in the shell do not exceed the moderate-rotation range the functionals presented here will reduce to those published in [9]. If, on the other hand, the rotations around the tangent to the middle surface are large, whereas those around the normal remain small, the Hu-Washizu functional derived here reduces to its simplified versions given in [28] for various degrees of accuracy of the strain energy function.

## 2. PRELIMINARY RELATIONS

The main purpose of this section is to outline in brief the relations governing behavior of a shell undergoing finite rotations but small strains within the range of the first-approximation theory[29-3 1] and to put them into the form of a nonlinear boundary-value problem with displacements as the basic independent variable. These relations were derived in detail in [12,20].

We shall clarify at first the calculus conventions adopted here. The composition of two second-order tensors S and T will be denoted by  $ST = S^{ij}T_{jk}g_{ij} \otimes g^{k}$ , whereas the full contraction by  $S \cdot T = S^{ij}T_{ij}$ . By  $1 = g_i \otimes g^i$  we shall denote the identity (unit) tensor. The symbols T', T', T' denote the transpose, the symmetric part and the antisymmetric (skew) part of the tensor T, respectively. We shall also employ the following operations: cross product of a vector v and a second-order tensor T, denoted  $v \times T =$  $v^i T^{ik}(\mathbf{g}_i \times \mathbf{g}_j) \otimes \mathbf{g}_k = -(\mathbf{T}' \times \mathbf{v})'$  with a second-order tensor as a result; cross product of two second-order tensors  $S \times T = S^{ij}T^{kl}g_i \otimes (g_j \times g_k) \otimes g_j$  with a third-order tensor as a result. In particular for the third-order Levi-Civita permutation tensor  $\mathbf{E} = e^{ijk} \mathbf{g}_i \otimes \mathbf{g}_j \otimes \mathbf{g}_k$ we have  $E = -1 \times 1$ . Finally, we recall that every second-order antisymmetric tensor  $T<sup>a</sup>$  has a unique representation in the form  $T^{\alpha} = v \times 1 = 1 \times v$ , where the axial vector v is equal to  $v = -\frac{1}{2}E \cdot T = -\frac{1}{2}e^{ijk}T_{jk}g_{ij}$ . The following sequence of operations should be observed: the cross product must be performed prior to the composition and full contraction. In other cases brackets will indicate the right order.

The shell  $\mathscr P$  in the undeformed state occupies a simply connected region in  $\mathscr R^3$ . Its middle surface  $M$  is a regular simply connected surface delimited by a picce-wise regular boundary contour  $\mathscr{C}$ . The vector  $\mathbf{r} = \mathbf{r}(\theta^{\alpha})$ ,  $\alpha = 1, 2$ , is the position vector of the surface  $\mathscr{M}$ and  $0^{\circ}$  is a curvilinear parametrization with the corresponding natural basis  $a_2 = r_{1\circ}$  (comma denotes differentiation).

In the sequel we shall consider differentiable tensor fields defined on  $\mathcal{M}$  and operators acting between them. Let  $\mathcal{V}_k$ ,  $k = 1, ..., N$ , and  $\mathcal{U}_k$  be any finite-dimensional Euclidean vector spaces and  $\mathcal{F} = \bigotimes_{k=1}^{N} \mathcal{V}_k$  be a tensor product space. Then the total differential of a possibly nonlinear function  $T = T(u)$ ,  $T \in \mathcal{T}$ ,  $u \in \mathcal{U}$ , is denoted by  $dT = T \otimes \nabla_u du$ ,  $d\mathbf{T} \in \mathcal{F}$ , du $\in \mathcal{U}$ , where () $\nabla_u = \partial f / \partial u'g'$  is the derivative operator and du = du'g<sub>i</sub>. The gradient  $\mathbf{T} \otimes \nabla_u = \frac{\partial \mathbf{T}}{\partial u^i} \otimes \mathbf{g}^i$  is a tensor in the space  $\mathscr{T} \otimes \mathscr{U}^*$ , where  $\mathscr{U}^*$  denotes the conjugate space of  $\mathcal{U}$ , and the partial differentiation with respect to  $\mathbf{u}^i$  should be applied to the product  $T = v_1 \otimes v_2 \otimes \cdots \otimes v_N$  according to the Leibniz rule. If, further,  $\mathcal{V}_N = \mathcal{U}^*$  and  $\dim \mathcal{U}^* = 3$  we define the divergence and curl as  $TV_u = (\partial T/\partial u')g'$  and  $T \times V_u = \partial T/\partial u'$   $\times g'$ , respectively. Finally, whenever the space  $\mathcal U$  is identical with  $T_M$ , i.e. a plane tangent to  $\mathcal M$ at a point  $M \in \mathcal{M}$ , we shall omit the subscript  $T_M$  in () $\nabla_{T_M}$  writing simply () $\nabla$ .

Now, let  $\xi = A(\theta)$ ,  $\theta \in \Theta$ ,  $\xi \in \Xi$ , be an operator acting between the Hilbert spaces  $\Theta$ and  $\Xi$  of tensor fields defined on the surface  $\mathcal{M}$ . Then  $\delta \xi = DA(\theta, \delta \xi)$  denotes its Gâteaux differential. However, further we shall abide by the traditional concepts referring to it as to the variation of the operator *A*. Whenever the operator  $DA$  is linear in  $\delta\theta$  we shall write  $\delta \zeta = (\delta A/\delta \theta) \delta \theta$ , where  $\delta A/\delta \theta$  denotes the Gâteaux derivative. For the precise mathematical background of the notions just introduced, see, e.g. the monographs by Vainberg[32, 33]. We assume that all functions and operators we shall deal with are differentiable up to the order needed.

Let us begin by introducing the fields inherent in the surface  $\mathcal{M}$ . These are  $\mathbf{a} = \mathbf{a}_{\alpha} \otimes \mathbf{a}^{\alpha}$ the metric tensor, **n**—the unit normal vector,  $\boldsymbol{\varepsilon} = \varepsilon^{a\beta} \boldsymbol{a}_a \otimes \boldsymbol{\alpha}_\beta$ —the surface permutation tensor,  $\mathbf{b} = -\mathbf{n} \otimes \nabla$ —the curvature tensor. Besides, v, t, n denote the Darboux orthonormal triad along the contour  $\mathscr C$  with t as the unit tangent vector.  $dA$  is the area element of the surface  $\mathcal{M}$ , whereas ds the length element of the contour  $\mathcal{C}$ . The following relations hold true on the surface  $\mathcal{M}$ :

$$
\mathbf{r} \otimes \mathbf{V} = \mathbf{a} = 1 - \mathbf{n} \otimes \mathbf{n}, \qquad \mathbf{\varepsilon} = -\mathbf{n} \times \mathbf{a} = -\mathbf{a} \times \mathbf{n} = -\mathbf{n} \times 1,
$$
  

$$
\mathbf{\varepsilon}\mathbf{\varepsilon} = -\mathbf{a}, \qquad \mathbf{\varepsilon} \cdot \mathbf{\varepsilon} = 2, \qquad \mathbf{n} = -\frac{1}{2} \mathbf{1} \times \mathbf{1} \cdot \mathbf{\varepsilon},
$$
  

$$
\mathbf{t} = \mathbf{r}_s, \qquad \mathbf{v} = \mathbf{t} \times \mathbf{n},
$$
  

$$
\mathrm{d}A = \mathbf{\varepsilon} \cdot (\mathrm{d}\mathbf{r}_1 \otimes \mathrm{d}\mathbf{r}_2), \qquad \mathrm{d}\mathbf{r}_1, \mathrm{d}\mathbf{r}_2 \in T_M.
$$
  
(2.1)

These quantities describe the geometry of the undeformed surface  $\mathcal{M}$ . We assume that they are known functions of the position vector r. Therefore the undeformed configuration of the shell will serve for the reference configuration and this means adoption ofthe Lagrangian description.

The shell  $\mathscr P$  subjected to the action of some external system of forces will deform to another configuration  $\mathscr P$  with the middle surface  $\mathscr M$  delimited by the contour  $\mathscr C$ . Quantities pertinent to the surface  $\vec{M}$  and equivalent to the ones already introduced on  $\vec{M}$  will be represented by the same symbols with the addition of overbars, i.e.  $\bar{r}$ ,  $\bar{a}$ ,  $\bar{n}$ , () $\bar{\nabla}$ , etc. With this modification the relations (2.1) still hold true on the surface  $\bar{M}$ . The linear transformation mapping dre  $T_M$  into dre  $\bar{T}_M$  is a two-point surface deformation gradient tensor

$$
\Gamma = \bar{\mathbf{r}} \otimes \mathbf{V} = \mathbf{a} + \mathbf{u} \otimes \mathbf{V},\tag{2.2}
$$

and the vector field  $\mathbf{u} = \mathbf{r} - \mathbf{r}$  is the displacement of the surface  $\mathcal{M}$ .

The following equations link the quantities in both configurations:

$$
J = \frac{d\overline{A}}{d\overline{A}} = \frac{1}{2}\Gamma'\overline{\epsilon}\Gamma \cdot \epsilon, \qquad \overline{\epsilon} = J^{-1}\Gamma\epsilon\Gamma',
$$
  
\n
$$
\overline{n} = -\frac{1}{2}J^{-1}\mathbf{1} \times \mathbf{1} \cdot \Gamma\epsilon\Gamma', \qquad (\mathbf{V}) = (\mathbf{V})\overline{\mathbf{V}}\Gamma,
$$
  
\n
$$
\overline{s} = a_t \, ds, \qquad a_t = |\overline{a}_t| = \sqrt{1 + 2\gamma_t}
$$
  
\n
$$
\overline{a}_t = \Gamma \mathbf{t} = a_t \overline{\mathbf{t}} = \mathbf{t} + \mathbf{u}_s,
$$
  
\n
$$
\overline{v} = \mathbf{R}v, \qquad \overline{\mathbf{t}} = \mathbf{R}t, \qquad \overline{\mathbf{n}} = \mathbf{R}\mathbf{n},
$$
  
\n
$$
\mathbf{R} = \overline{\mathbf{t}} \times \overline{\mathbf{n}} \otimes v + \overline{\mathbf{t}} \otimes t + \overline{\mathbf{n}} \otimes \mathbf{n} = \mathbf{e} \otimes \mathbf{e} + \cos \alpha (1 - \mathbf{e} \otimes \mathbf{e}) + \sin \alpha \mathbf{e} \times \mathbf{1},
$$
  
\n
$$
\sin \alpha \mathbf{e} = v \times \overline{v} + \mathbf{t} \times \overline{\mathbf{t}} + \mathbf{n} \times \overline{\mathbf{n}},
$$
  
\n
$$
2 \cos \alpha + 1 = v \cdot \overline{v} + \mathbf{t} \cdot \overline{\mathbf{t}} + \mathbf{n} \cdot \overline{\mathbf{n}}.
$$

The orthogonal tensor  $\bf{R}$  in (2.3) describes the total rotation of the boundary element by the angle  $\alpha$  around the axis e and  $a_i$ , is the stretch of this element.

The relations  $(2.1)$ - $(2.3)$  depict the geometric aspect of the theory. When substituted to the surface deformation measures (for justification of the measures assumed see [12, 20]): the surface strain tensor  $\gamma$  and the change of curvature tensor  $\chi$ ,

$$
\gamma = \frac{1}{2}(\Gamma'\Gamma - a), \quad \chi = J\Gamma'(\mathbf{\tilde{n}} \otimes \nabla) + \mathbf{b}(1 + a \cdot \gamma) \tag{2.4}
$$

they yield the kinematic relations

$$
\gamma(\mathbf{u}) = (\mathbf{u} \otimes \nabla)^{t} + \frac{1}{2} (\mathbf{u} \otimes \nabla)^{t} (\mathbf{u} \otimes \nabla),
$$
  

$$
\chi(\mathbf{u}) = \mathbf{m} \otimes \nabla + (\mathbf{u} \otimes \nabla)^{t} (\mathbf{m} \otimes \nabla) + \mathbf{b} [1 + \mathbf{a} \cdot \gamma(\mathbf{u})],
$$
 (2.5)

where

$$
\mathbf{m}(\mathbf{u}) = J\,\mathbf{\bar{n}} = -\frac{1}{2}\mathbf{1} \times \mathbf{1} \cdot \Gamma \boldsymbol{\varepsilon} \Gamma' = \frac{1}{2} \boldsymbol{\varepsilon}^{\alpha \beta} (\mathbf{a}_{\alpha} + \mathbf{u}_{,\alpha}) \times (\mathbf{a}_{\beta} + \mathbf{u}_{,\beta}). \tag{2.6}
$$

The other aspect of the problem is related to the phenomena resulting from the action of external forces on the shell. These forces depend in general on the actual configuration  $\overline{\mathscr{P}}$ . Let us assume that the contour  $\mathscr{C}$  consists of two separate parts:  $\mathscr{C}_{f}$ -along which the force  $\mathbf{B} = \mathbf{B}(\mathbf{u})$  and the moment  $\mathbf{k} = \mathbf{k}(\alpha, \mathbf{e})$  per unit length are prescribed known functions of the displacement u, the angle of the total rotation  $\alpha$  and the axis e, respectively; and its completion  $\mathscr{C}_\nu$  with a fixed configuration of the shell boundary. Similar division can be made on the middle surface  $M$ . However, we neglect in further considerations the case wherein some parts of  $\mathcal M$  have prescribed geometry, because it leads to problems with multiply-connected regions and seems of little practical importance. Thus we assume that the whole surface  $\mathcal M$  is subject to the load  $p = p(u)$  per unit area. Now we can impose the equilibrium condition in the global form, demanding the principle of virtual work to hold true

$$
\iint_{\mathscr{M}} (\mathbf{N} \cdot \delta \mathbf{y} + \mathbf{M} \cdot \delta \mathbf{y}) dA = \iint_{\mathscr{M}} \mathbf{p} \cdot \delta \mathbf{u} dA + \int_{\mathscr{C}_{\ell}} (\mathbf{B} \cdot \delta \mathbf{u} + \mathbf{k} \cdot \delta \omega) ds,
$$
  

$$
\mathbf{p} = \mathscr{F} \mathbf{\tilde{p}}, \quad \mathbf{B} = a_{\ell} \mathbf{\tilde{B}}, \quad \mathbf{k} = a_{\ell} \mathbf{\tilde{k}}.
$$
 (2.7)

Here N and M denote the Lagrangian internal force and moment tensors—the counterparts of the Kirchhoff (or the second Piola-Kirchhoff) stress tensor in the three-dimensional theory of deformable continuum[34]. For an alternative approach, based on the Jaumanntype of internal forces, see [21-23, 34].

Note the occurrence of the term  $\delta \omega = \delta \omega i$  in (2.7). It represents a small incremental rotation of the boundary element in the actual state by an infinitesimal angle  $\delta\omega$  around the axis i. Note also that in general  $\delta\omega$  does not coincide with  $\delta(\alpha e)$ . The equations linking these terms and the boundary configuration variables can be derived from the condition for the spin tensor  $\delta\omega \times 1 = \dot{\mathbf{R}}' \delta \mathbf{R} [17]$  and eqns  $(2.3)_{6-8}$ . We have

$$
\delta \omega = \frac{1}{a_t} \mathbf{t} \times \mathbf{R}' \delta \mathbf{\tilde{a}}_t + \mathbf{t} (\mathbf{t} \cdot \delta \omega) = \frac{\sin \alpha}{\alpha} \mathbf{1} + \frac{\alpha - \sin \alpha}{\alpha} \mathbf{e} \otimes \mathbf{e} + \frac{1 - \cos \alpha}{\alpha} \mathbf{e} \times \mathbf{1} \delta(\alpha \mathbf{e}), \qquad (2.8)
$$

$$
\mathbf{t} \cdot \delta \omega = \mathbf{\tilde{v}} \cdot \delta \mathbf{\tilde{n}}, \qquad \delta \mathbf{\tilde{a}}_t = \delta \mathbf{u}_s.
$$

Besides, from the identities  $\mathbf{\vec{n}}^2 = 1$  and  $\mathbf{\vec{n}} \cdot \mathbf{\vec{a}}_i = 0$  and  $(2.8)$ , we can derive the following expression for  $\delta\bar{n}$  at the boundary

$$
\delta \mathbf{\bar{n}} = (\mathbf{t} \cdot \delta \boldsymbol{\omega}) \mathbf{\bar{v}} - \frac{1}{a_t} \mathbf{\bar{t}} (\mathbf{\bar{n}} \cdot \delta \mathbf{u}_{,t}). \tag{2.9}
$$

Having substituted (2.5), (2.8), and (2.9) to (2.7) and applying the Stokes theorem, the principle of virtual work can be transformed to the equivalent form

$$
-\iiint_{\mathscr{M}} (\mathbf{TV} + \mathbf{p}) \cdot \delta \mathbf{u} dA + \int_{\mathscr{C}_{\ell}} \{ [\mathbf{TV} - \mathbf{B} + [(\mathbf{F} + \mathbf{t} \times \mathbf{k}) \mathbf{R}^{\prime}]_{,s}] \cdot \delta \mathbf{u} + (J\bar{\mathbf{v}} \mathbf{\Gamma} \mathbf{M} \mathbf{v} + \mathbf{k} \cdot \mathbf{t}) \mathbf{t} \cdot \delta \omega \} dS - \sum_{i=1}^{N} (\mathbf{F}_{i} + \mathbf{t}_{i} \times \mathbf{k}_{i}) \mathbf{R}^{\prime} \cdot \delta \mathbf{u}_{i} = 0 \quad (2.10)
$$

and the symbols used in (2.10) represent

$$
\mathbf{T} = \Gamma[\mathbf{N} + (\mathbf{M} \cdot \mathbf{b})\mathbf{a}] + (\mathbf{m} \otimes \nabla)\mathbf{M} - [(\mathbf{\Gamma}\mathbf{M})\nabla] \times \mathbf{\Gamma}\mathbf{\varepsilon}
$$

$$
\mathbf{F} = \left(\frac{1}{a_i} J \mathbf{\tilde{t}} \mathbf{\Gamma} \mathbf{M} \mathbf{v}\right) \mathbf{n} \tag{2.11}
$$

$$
\mathbf{F}_i = \mathbf{F}(s - 0) - \mathbf{F}(s + 0).
$$

The condition  $(2.10)$  yields directly the local form of the Lagrangian equilibrium conditions:

the equilibrium equations on  $\mathcal M$ 

$$
TV + p = 0, \tag{2.12}
$$

the natural boundary conditions along  $\mathscr{C}_f$ 

$$
\mathbf{Tv} - \mathbf{B} + [(\mathbf{F} + \mathbf{t} \times \mathbf{k})\mathbf{R}^{\mathsf{T}}]_{,s} = \mathbf{0},
$$
  

$$
J\bar{\mathbf{v}}\mathbf{F}\mathbf{M}\mathbf{v} + \mathbf{k} \cdot \mathbf{t} = 0,
$$
 (2.13)

and the jump conditions

$$
\mathbf{F}_i + \mathbf{t}_i \times \mathbf{k}_i = \mathbf{0} \tag{2.14}
$$

at the points  $M_i \in \mathcal{C}_f$ ,  $i = 1, ..., N$ , at which either the moment k is discontinuous or the contour  $\mathscr{C}_f$  has corners.

It remains to impose the geometric boundary conditions. This consists in prescribing the position of the deformed boundary contour  $\mathscr{C}_u$  and the orientation of its Darboux

### 1166 M. L. Szwabowicz

orthonormal triad,  $\bar{v}^*$ ,  $\bar{t}^*$ ,  $\bar{n}^*$ . However, in virtue of eqns (2.3)<sub>16</sub>, it suffices to specify merely the orientation of the unit normal  $\bar{n}^*$ . Yet  $\bar{n}$  itself must still satisfy the kinematic constraint  $\bar{\mathbf{n}} = \bar{\mathbf{t}} \times (\bar{\mathbf{n}} \times \bar{\mathbf{t}})$ . Consequently,  $\bar{\mathbf{n}}$  is a function of  $\bar{\mathbf{t}}$  and some scalar parameter  $\phi$ , i.e.  $\mathbf{\bar{n}} = f(\mathbf{\bar{t}}, \phi)$  and, thus, so is the rotation tensor  $\mathbf{R} = \mathbf{R}(\mathbf{\bar{t}}, \phi)$ . This remark also concerns the static boundary conditions  $(2.13)$ , where **R** appears explicitly and implicitly in the function  $k = k(\alpha, e)$ . However, it is not clear up to now whether there exists any standard form of such a function. In [12, 17–20], where the parameter  $\phi$  was assumed to be  $n<sub>v</sub> = \mathbf{\bar{n}} \cdot \mathbf{v}$ , the resulting expression for  $\bar{n}(\bar{t}, n_s)$  turned out to be nonunique. The same happens when  $\phi$ is identified with the angle  $\alpha$  of the total boundary rotation. To avoid these difficulties we impose the conditions valid for any parameter used:

$$
\mathbf{u} - \mathbf{u}^*(s) = 0,
$$
  
\n
$$
\mathbf{\bar{n}} \cdot \mathbf{\bar{t}}^* \times \mathbf{n}^*(\mathbf{\bar{t}}^*, \phi^*) = 0 \quad \text{along } \mathscr{C}_u,
$$
  
\n
$$
\mathbf{u}_k - \mathbf{u}_k^*(s_k) = 0, \qquad k = 1, ..., K, \text{ at } M_k \in \mathscr{C}_u.
$$
\n(2.15)

In (2.15)<sub>2</sub> the expression  $\bar{\mathbf{t}}^* = (1/a_t^*)(\mathbf{t} + \mathbf{u}_{ss}^*)$  should be substituted for  $\bar{\mathbf{t}}^*$ . The condition  $(2.15)$ , prescribes the displacements at the corners of  $C<sub>u</sub>$ .

The equations displayed in this section form a system valid for arbitrary deformations of the surface  $M$ . They comprise two independent kinds of variables, namely variables of geometric and static character. To bind this system up into a complete boundary-value problem we still need the constitutive equations.

## 3. THE POTENTIALS AND THE INITIAL FUNCTIONAL

Apart from interpreting the principle of virtual work (2.7) as a balance condition, it can also be viewed as a condition equating to zero a Gateaux differential

$$
\delta I = \iint_{\mathscr{M}} (\mathbf{N} \cdot \delta \gamma + \mathbf{M} \cdot \delta \chi - \mathbf{p} \cdot \delta \mathbf{u}) dA - \int_{\mathscr{C}_{f}} (\mathbf{B} \cdot \delta \mathbf{u} + \mathbf{k} \cdot \delta \omega) ds \tag{3.1}
$$

of some unknown functional I with the kinematic relations as constraints. The question, whether there really exists such a functional, is equivalent to the problem of finding potentials for the internal forces  $N$ ,  $M$  and the external forces  $p$ ,  $B$ ,  $k$  treated now as operators acting from the spaces of geometric variables to the conjugate spaces of forces. General concepts concerning the potentiality of operators are broadly discussed in [32, 33] and with specification to mechanics in [26, 34]. We only recall that if  $DA(\theta, \delta\theta)$ ,  $\theta \in \Theta$ , exists and is hemicontinuous in  $\theta$  and the domain  $\Theta$  is convex, then the condition for potentiality demands the equality

$$
\langle DA(\theta, \delta\theta_1), \delta\theta_2 \rangle = \langle DA(\theta, \delta\theta_2), \delta\theta_1 \rangle \tag{3.2}
$$

to hold true. The corresponding functional follows from the formula

$$
F(\theta) = \int_0^1 \langle A(\tau \theta), \theta \rangle d\tau.
$$
 (3.3)

Let us notice that according to  $(3.1)$  the potential of the internal forces N, M (or the internal energy function) can only depend on the variables  $\gamma$  and  $\gamma$ . Formally this means that the internal energy of the shell is determined solely by the deformation of the middle surface  $M$ . This confines applicability of the theory to those deformations, in which the principal strains remain small in comparison with unity everywhere in the shell. Otherwise (3.1) ought to include additional variables describing the deformation across the thickness *h* of the shall. Besides, let us consider only the elastic deformations. These restrictions coincide well with the assumptions of the so-called first-approximation theory developed

in [29-31]. Profiting directly from the results contained therein we may postulate the existence of a two-dimensional strain energy density function defined per unit area of  $M$ in the approximate form

$$
\Sigma = \frac{h}{2} \mathbf{H} \cdot \left( \gamma \otimes \gamma + \frac{h^2}{12} \chi \otimes \chi \right), \tag{3.4}
$$

where H denotes the fourth-order modified elasticity tensor[31]. Taking the variation of (3.4) we can obtain the constitutive equations in the form

$$
N = \frac{\delta \Sigma}{\delta \gamma} = hH \cdot \gamma, \qquad M = \frac{\delta \Sigma}{\delta \chi} = \frac{h^3}{12} H \cdot \chi \tag{3.5}
$$

typical for the linearly elastic materials.

As it seems impossible to characterize any universal class of potential loads p and B, we assume that the corresponding potentials  $\Pi(u)$  and  $\Phi(u)$  exist, so that the condition (3.2) is satisfied. For the particular case of the pressure load we refer the reader to [35-37]. Yet we shall concentrate on the question of the potentiality of the moment k. This problem was solved by Simmonds[38] for a rigid body in rotational motion in terms of the Rodrigues vector  $\beta = 2 \tan (\alpha/2)$ e. This variable becomes singular any time  $\alpha = n\pi$ . We present here an alternative solution, free of this disadvantage.

Let us point out that the main difficulty here is due to the fact that the infinitesimal rotation  $\delta \omega$  is not a Gâteaux differential of any vector field. To overcome this we must modify the expression for the work rate  $\delta\Psi = \mathbf{k} \cdot \delta\omega$  with the aid of the relation (2.8), which substituted for  $\delta\omega$  yields

$$
\delta \Psi = \mathbf{k} \cdot \delta \omega = \hat{\mathbf{k}} \cdot \delta \alpha, \tag{3.6}
$$

where the pseudo moment

$$
\mathbf{\hat{k}} = \frac{\alpha - \sin \alpha}{\alpha^3} \alpha \times (\alpha \times \mathbf{k}) + \frac{\cos \alpha - 1}{\alpha^2} \alpha \times \mathbf{k} + \mathbf{k}
$$
 (3.7)

belongs to a vector space conjugate to another space, whose members are the vectors  $\alpha = \alpha e$ . Further we may apply the routine methods involving (3.2), which eventually results in the condition for the potentiality of the moment in the local form

$$
\mathbf{\hat{k}} \times \nabla_{\alpha} = \mathbf{0}.\tag{3.8}
$$

Some important conclusions may be deduced from (3.8):

(i) the constancy of the moment k, i.e.  $k \times \nabla_{\alpha} = 0$ , does not suffice to guarantee the existence of a potential;

(ii) if  $k = ke$  with  $k = const$  the potential takes the simple form  $\Psi(\alpha) = k\alpha$ ; moreover, if  $e = t$ , then *k* is the bending moment and  $\alpha$  is the winding angle of the shell around the contour  $\mathscr{C};$ 

(iii) if  $\mathbf{k} = \mathbf{\bar{n}} \times \mathbf{H}$  with  $\mathbf{H} = \text{const}$ , then a potential in the form  $\Psi(\mathbf{\bar{n}}) = \mathbf{H} \cdot (\mathbf{\bar{n}} - \mathbf{n})$  exists and H corresponds to the resultant static moment of dead-load type tractions acting on the shell lateral boundary surface  $\partial P$  (see [17]).

There is still one important aspect, concerning operators and the related potentials, closely connected with the complementary variational formulations. If an operator turns out to be strictly monotone and potential, then the corresponding potential is strictly convex[33]. The converse is also true. On the other hand, strict monotonicity implies invertibility of the operator-a property enabling effectuation of the Legendre involuntary transformation[24-26]. Since the strain energy density function is strictly convex by definition, there always exists such fourth-order tensor **E** that

$$
\gamma = \frac{1}{h} \tilde{\mathbf{E}} \cdot \mathbf{N}, \qquad \chi = \frac{12}{h^3} \tilde{\mathbf{E}} \cdot \mathbf{M}. \tag{3.9}
$$

From these inverted constitutive equations follows the so-called first complementary energy density function<sup>[34]</sup>

$$
\Sigma^{c}(\mathbf{N},\mathbf{M})=\mathbf{N}\cdot\boldsymbol{\gamma}+\mathbf{M}\cdot\boldsymbol{\chi}-\Sigma(\boldsymbol{\gamma},\boldsymbol{\chi})=\frac{1}{2h}\tilde{\mathbf{E}}\cdot\left(\mathbf{N}\otimes\mathbf{N}+\frac{12}{h^{2}}\mathbf{M}\otimes\mathbf{M}\right),\quad(3.10)
$$

which, by virtue of the properties of the Legendre transformation (3.10), satisfies the equations

$$
\gamma = \frac{\delta \Sigma^c}{\delta N}, \qquad \chi = \frac{\delta \Sigma^c}{\delta M}.
$$
 (3.11)

The same procedure may be applied to the potentials  $\Pi(u)$  and  $\Phi(u)$ , provided the inverse operators  $\mathbf{u} = \mathbf{u}(\mathbf{p})$  and  $\mathbf{u} = \mathbf{u}(\mathbf{B})$  exist. This would result in the following relations:

$$
\Pi^c(\mathbf{p}) = \mathbf{p} \cdot \mathbf{u} - \Pi(\mathbf{u}), \qquad \delta \Pi^c = \mathbf{u} \cdot \delta \mathbf{p},
$$
  
\n
$$
\Phi^c(\mathbf{B}) = \mathbf{B} \cdot \mathbf{u} - \Phi(\mathbf{u}), \qquad \delta \Phi^c = \mathbf{u} \cdot \delta \mathbf{B}.
$$
\n(3.12)

The quantities  $\Pi^c$  and  $\Phi^c$  should be termed complementary work of the external forces p andB.

Before passing to the direct examination of the functionals and variational principles, it seems worthwhile to expose a general view of the nonlinear boundary-value problem we have to deal with. A system of the following equations constitutes this problem:

(i) kinematic relations  $(2.5)$ :  $\varepsilon = \varepsilon(u)$ ;

(ii) equilibrium equations on  $\mathcal{M}(2.11)$ , (2.12):  $\mathbf{T}(u, \sigma)\nabla + \mathbf{p} = \mathbf{0}$ ;

(iii) static boundary conditions along  $\mathscr{C}_f$  (2.13), (2.14) :  $f(u_0, \sigma_0, \sigma_0^*) = 0$ ;

(iv) constitutive equations (3.5):  $\sigma = \sigma(\varepsilon)$ , and the inverted constitutive equations  $(3.9)$ :  $\varepsilon = \varepsilon(\sigma)$ ;

(v) geometric boundary conditions along  $\mathcal{C}_{\mu}$  (2.15):  $g(u_{i_{c}}, u_{i_{c}}^{*}) = 0$ .

Three kinds of fields occur in the equations (i)–(v): the displacements  $u = \{u \text{ on } \mathcal{M}\}\$ ;  $\mathbf{u}_{i\sigma}$ , ae along  $\mathscr{C}$ , the strain measures  $\varepsilon = \{\gamma, \chi\}$  and the conjugate stress measures  $\sigma = \{N, M\}$ . There are also parameters: fixed displacements along  $C_u u_x^* = {u^*, \alpha^* e^*}$  and prescribed functions  $\mathbf{p} = \mathbf{p}(\mathbf{u})$  on *M* and  $\sigma_k^*(u_k) = {\mathbf{B}, \mathbf{k}}$  along  $\mathcal{C}_f$  specifying the loads with respect to the actual displacements. All conceivable fields  $[u, \varepsilon, \sigma]$  form the space of states  $\mathscr S$  of the shell  $\mathscr P$  and the equations (i)-(v) determine these particular points  $[u_0, \varepsilon_0, \sigma_0] \in \mathscr S$ , which are the solutions to the boundary-value problem.

Resorting to the origins of this section, let us point out that the static equations (ii) and (iii) resulted from the variational principle  $\delta I = 0$  after substitution of the kinematic relations (i) to (3.1). Thus, if the functional

$$
I[u,\varepsilon] = \iint_{\mathscr{U}} \left[ \Sigma(\varepsilon) - \Pi(u) \right] dA - \int_{\varepsilon_f} \left[ \Phi(u) + \Psi(\alpha \mathbf{e}) \right] ds \tag{3.13}
$$

exists, the boundary-value problem  $(i)-(v)$ , may be converted into an equivalent constrainted variational problem:

find 
$$
[u_0, \varepsilon_0, \sigma_0 = \sigma(\varepsilon_0)] \in \mathcal{S}
$$
 such that  $I[u_0, \varepsilon_0] =$  stationary values of  $I[u, \varepsilon]$ . (3.14)  
\n
$$
[u, \varepsilon] \in \mathcal{D}
$$

Variational formulation in the geometrically nonlinear thin elastic shell theory 1169

The subset  $\mathscr{D} \subset \mathscr{S}$ 

$$
\mathscr{D} = \{ [u, \varepsilon] \, | \, \varepsilon - \varepsilon(u) = 0, g(u_{\varepsilon}, u^*) = 0 \}
$$

is the set of all feasible strain and displacement fields on  $\mathcal{M}$ .

## 4. GENERALIZED VARIATIONAL PRINCIPLES

The functional (3.13) must be supplemented by subsidiary conditions to acquire a definite sense. Generally, such a situation is inconvenient in practical applications, where the unconstrained (or free) formulations are preferable. Therefore in this section we shall be mainly preoccupied with transformations leading to free functionals of the Hu-Washizu type. To this end we shall profit from the well-known techniques developed by Courant and Hilbert[24] and Lanczos[25]. The books[1-3, 26] lay particular stress on the application of these ideas to mechanics. Figure 1 illustrates the procedure followed.

Let us introduce the kinematic relations (i) and the geometric boundary conditions (v) into the functional  $I$  via the Lagrange multiplier rule. This operation spreads the domain of the resulting functional  $I_1$  on the whole space  $\mathcal S$ 

$$
I_1[u, \varepsilon, \sigma] = \iint_{\mathscr{M}} {\left\{ \Sigma - \Pi - \mathbf{N} \cdot [\gamma - \gamma(\mathbf{u})] - \mathbf{M} \cdot [\chi - \chi(\mathbf{u})] \right\} dA - \int_{\mathscr{C}_r} {\left[ \Phi + \Psi \right] ds - \int_{\mathscr{C}_r} {\left[ \mathbf{P} \cdot (\mathbf{u} - \mathbf{u}^*) + M \mathbf{n} \cdot \mathbf{t}^* \times \mathbf{n}^* \right] ds - \sum_{k=1}^K \mathbf{G}_k \cdot (\mathbf{u}_k - \mathbf{u}_k^*), \quad (4.1)}
$$

for the fields of Lagrange multipliers, N, M pertaining to (i) and  $P$ ,  $M$ ,  $G<sub>k</sub>$  pertaining to (v), must belong to the conjugate spaces of forces. Direct examination of the variation of  $I_1$  affirms this statement. Following the same pattern that led from (2.7) to (2.10), with particular emphasis on the fact that the geometry of  $\mathscr{C}_u$  is already determined, we can derive  $\delta I_1$  in the form

$$
\delta I_{1} = \iiint_{\mathscr{A}} \left\{ \left( \frac{\delta \Sigma}{\delta \gamma} - N \right) \cdot \delta \gamma + \left( \frac{\delta \Sigma}{\delta \chi} - M \right) \cdot \delta \chi \right.\n- [\gamma - \gamma(u)] \cdot \delta N - [\chi - \chi(u)] \cdot \delta M - [TV + p] \cdot \delta u \right\} dA\n+ \int_{\mathscr{G}_{f}} \left\{ [Tv - B + [(F + t \times k)R']_{\nu}]\cdot \delta u \right.\n+ (J\bar{v} \Gamma M v + k \cdot t)t \cdot \delta \omega \right\} ds - \sum_{i=1}^{N} (F_{i} + t_{i} \times k_{i})R' \cdot \delta u_{i}\n- \int_{\mathscr{G}_{\nu}} [(u - u^{*}) \cdot \delta P + n \cdot t^{*} \times n^{*} \delta M] ds - \sum_{k=1}^{K} (u_{k} - u_{k}^{*}) \cdot \delta g_{k}\n+ \int_{\mathscr{G}_{\nu}} \left\{ [T^{*} v + (F^{*} R^{*})_{\nu} - P] \cdot \delta u + (\bar{v}^{*} \Gamma M v - M) t \cdot \delta \omega \right\} ds\n- \sum_{k=1}^{K} (F_{k}^{*} R^{*'} + G_{k}) \cdot \delta u_{k}, \qquad (4.2)
$$

1170 M. L SZWABOWICZ





where  $F^* = F(u^*, \sigma_{1c})$  and  $T^* = T(u^*, \sigma_{1c})$ . The set of the stationarity conditions for the functional  $I_1$  contains not only the full system (i)–(v), but also includes relations identifying the Lagrange multipliers  $P$ ,  $M$ ,  $G_k$ , as the effective boundary force and bending moment appearing along  $\mathcal{C}_{\mu}$  due to the imposition of the geometric constraints (v) there. These relations placed in the last lines of (4.2) can serve to eliminate the multipliers P, M,  $G_k$ from  $I_1$  and so to unify its structure. Their substitution to (4.1) results in the following change of the boundary part  $I_b$  of the functional  $I_1$ :

$$
I_b[u_{\epsilon}, \sigma_{\epsilon}] = \int_{\mathcal{C}_f} [\Phi + \Psi] \, \mathrm{d} s + \int_{\mathcal{C}_v} \{ [\mathbf{T}^* \mathbf{v} + (\mathbf{F}^* \mathbf{R}^{*t})_{\mathbf{v}}] \cdot (\mathbf{u} - \mathbf{u}^*) + (J \bar{\mathbf{v}}^* \mathbf{\Gamma} \mathbf{M} \mathbf{v}) \tilde{\mathbf{n}} \cdot \tilde{\mathbf{v}}^* \} \, \mathrm{d} s - \sum_{k=1}^K \mathbf{F}_k^* \mathbf{R}^{*t} \cdot (\mathbf{u}_k - \mathbf{u}_k^*). \tag{4.3}
$$

Similar substitution to the expression for the variation (4.2) results in vanishing of the last lines, leaving, in consequence, only the relations (i)–(v) as the stationarity conditions of  $I_1$ .

Now the constraints included in the initial problem (3.14) may be removed and the problem itself can be restated as:

find 
$$
[u_0, \varepsilon_0, \sigma_0] \in \mathcal{S}
$$
 such that  $\delta I_1[u_0, \varepsilon_0, \sigma_0] = 0.$  (4.4)

This is the Hu-Washizu variational principle.

The functional  $I_1$  comprises explicitly the relations of the geometric type (i) and (v). The question arises whether it is possible to effect any transformation that would promote explicit occurrence of the static relations (ii) and (iii). Let us point out that the following equation holds true:

$$
N \cdot \gamma(u) + M \cdot \chi(u) = [uT + uQ - n\Gamma M v] \cdot \nabla - u \cdot T\nabla - u \cdot Q\nabla, \qquad (4.5)
$$

where

$$
Q=\tfrac{1}{2}(u\otimes\nabla)[N+(M\cdot b)a]-[(m-n)\otimes\nabla]M+\tfrac{1}{2}[(TM)\nabla]\times(u\otimes\nabla)\epsilon.
$$

Substituting this result to (4.1) and applying the Stokes theorem to the terms under the

divergence operator [ ]V we can derive an equivalent form of the functional  $I_1$ :

$$
J_1[u, c, \sigma] = -\iint_{\mathscr{M}} \left[ (\mathbf{N} \cdot \gamma + \mathbf{M} \cdot \chi - \Sigma) + (\mathbf{TV} + \mathbf{p}) \cdot \mathbf{u} + \mathbf{u} \cdot \mathbf{Q} \mathbf{V} + (\mathbf{\Pi} - \mathbf{p} \cdot \mathbf{u}) \right] dA + J_b[u_{\mathfrak{g}}, \sigma_{\mathfrak{g}}],
$$
\n(4.6)

where the boundary part  $J<sub>b</sub>$  is

$$
J_b[u_{\vert c}, \sigma_{\vert c}] = \int_{\mathscr{C}_f} \{ [\mathbf{T}v - \mathbf{B} + (\mathbf{F}\mathbf{R}'),_{s}] \cdot \mathbf{u} - (\Phi - \mathbf{B} \cdot \mathbf{u})
$$
  
+ (\mathbf{Q}v) \cdot \mathbf{u} - \mathbf{F}\mathbf{R}' \cdot \mathbf{t} - \mathbf{n} \mathbf{\Gamma} \mathbf{M}v - \Psi \} ds - \sum\_{i=1}^{N} \mathbf{F}\_i \mathbf{R}' \cdot \mathbf{u}\_i  
+ \int\_{\mathscr{C}\_v} \{ [\mathbf{T}^\*v + (\mathbf{F}^\* \mathbf{R}^{\*t})\_{,s}] \cdot \mathbf{u} + (\mathbf{Q}^\*v) \cdot \mathbf{u} - \mathbf{F}^\* \mathbf{R}^{\*t} \cdot \mathbf{t}   
- \mathbf{n} \mathbf{\Gamma} \mathbf{M}v - (\mathscr{F} \bar{v}^\* \mathbf{\Gamma} \mathbf{M} v) \tilde{\mathbf{n}} \cdot \tilde{v}^\* \} dS + \sum\_{k=1}^{K} \mathbf{F}\_k^\* \mathbf{r}^{\*t} \cdot \mathbf{u}\_k. (4.7)

Let us note some specific features of the functional  $J_i$ . Owing to the equivalence between  $I_1$  and  $J_1$  the variations of both functionals are equal,  $\delta I_1 = \delta J_1$ , and hence their stationarity conditions are the same. The first three terms under the surface integral in (4.6) are equal in value to the complementary energy function  $\Sigma^c$  (3.9). Besides, expressions equivalent to the complementary work  $\Pi^c$  and  $\Phi^c$  (3.12) have appeared under the surface and the  $\mathscr{C}_f$ -boundary integrals. Due to strong interference of rotations and arising hence nonlinearities the equilibrium equations (iii) could be separated in the surface integrand at the cost ofincluding a new term Q, having the character of residual internal forces. Due to the same reason it seems virtually impossible to handle the boundary integrands in a similar way.

A functional of this type, termed the total complementary energy functional, was brought out in [9, 13-16]. Thus the variational principle  $\delta J_1 = 0$  may be called the principle of generalized complementary energy.

# 5. DERIVED VARIATIONAL PRINCIPLES

In the original boundary-value problem (i)–(v) the fields  $\varepsilon$  and  $\sigma$  can be eliminated via substitution of the kinematic relations (i) and the constitutive equations (iv) to the static relations (ii) and (iii), thus gradually reducing the space of states  $\mathcal{S}$ . This process has its reflection in the variational approach.

Both functionals derived in the previous section,  $I_1$  and  $J_1$ , have the same stationarity conditions that result from (4.2). Adding them to these functionals in the form of supplementary constraints and then eliminating with their help some of the variable fields does not affect the stationarity conditions of the resulting reduced functionals. We shall apply this procedure to obtain six subsequent free variational principles.

Let us eliminate the strains  $\varepsilon$  from the functionals  $I_1$  and  $J_1$  by means of the inverted constitutive equations  $\varepsilon = \varepsilon(\sigma)$  (3.9). As a result we obtain two free functionals

$$
I_2[u,\sigma] = \iint_{\mathcal{M}} \left( -\Sigma^c(\sigma) - \Pi + \mathbf{N} \cdot \gamma(\mathbf{u}) + \mathbf{M} \cdot \chi(\mathbf{u}) \right) dA - I_b[u_c, \sigma_{ic}], \tag{5.1}
$$

$$
J_2[u,\sigma] = -\iint_{\mathscr{M}} \left[ \Sigma^c(\sigma) + (\mathbf{TV} + \mathbf{p}) \cdot \mathbf{u} + \mathbf{u} \cdot \mathbf{Q} \mathbf{V} + (\Pi - \mathbf{p} \cdot \mathbf{u}) \right] dA + J_b[u_{\vert c}, \sigma_{\vert c}]. \tag{5.2}
$$

Note that the equality  $\delta I_2 = \delta J_2$  is still preserved. To obtain the expression for  $\delta I_2$  one should replace the first two lines in (4.2) under the surface integral, i.e. the ones containing the constitutive equations and the kinematic relations, with the relations of the form  $\left[\varepsilon(\sigma) - \varepsilon(u)\right] \delta \sigma.$ 

The sense of the operation just completed consists in reducing the space  $\mathscr S$  to the space  $\mathcal{S}_R$  of the pairs [u,  $\sigma$ ] so that we may replace the problem (4.3) by a modified unconstrained problem:

find 
$$
[u_0, \sigma_0] \in \mathcal{S}_R
$$
 such that  $\delta I_2[u_0, \sigma_0] = 0$  or  $\delta J_2[u_0, \sigma_0] = 0.$  (5.3)

This is the Hellinger-Reissner variational principle.

A similar operation, based on the elimination of the stress field  $\sigma$  via the constitutive equations in their direct form  $\sigma = \sigma(\varepsilon)$  (3.5), leads to another pair of free functionals:

$$
I_{6}[u,\varepsilon] = -\iint_{\mathcal{M}}\left\{\Sigma(\varepsilon) + \Pi - hH \cdot \left[\gamma \otimes \gamma(u) + \frac{h^{2}}{12}\chi \otimes \chi(u)\right]\right\} dA - I_{b}[u_{\varepsilon}, \sigma(\varepsilon_{\varepsilon})],
$$
\n(5.4)

$$
J_{6}[u,\varepsilon] = -\iint_{\mathcal{M}} \left\{ \Sigma(\varepsilon) + \left[ \mathbf{T}(u,\sigma(\varepsilon))\nabla + \mathbf{p} \right] \cdot \mathbf{u} + \mathbf{u} \cdot \mathbf{Q}(u,\sigma(\varepsilon))\nabla + (\Pi - \mathbf{p} \cdot \mathbf{u}) \right\} dA + J_{b}[u_{\varepsilon},\sigma(\varepsilon_{\varepsilon})]. \tag{5.5}
$$

In the derivation of  $J_6$  use has been made of the relation

$$
2\Sigma(\varepsilon) = \mathbf{N}(\gamma) \cdot \gamma + \mathbf{M}(\gamma) \cdot \gamma. \tag{5.6}
$$

Again the equality  $\delta I_6 = \delta J_6$  holds true and to obtain the variation of these functionals the first line in (4.2), now vanishing identically, should be removed, and elsewhere the righthand sides of (3.5) should be substituted for N and M. The functionals  $I_6$  and  $J_6$  are accompanied with a variational principle similar to (5.3).

Carrying this process on we can eventually derive a pair of free functionals depending solely on the displacement field *u*. These two functionals result either from  $I_6$  and  $J_6$  via substitution of the kinematic relations  $\varepsilon = \varepsilon(u)$ , or from  $I_2$  and  $J_2$  via substitution of the compound relation  $\sigma = \sigma(\varepsilon(u))$  obtained by introduction of the kinematic relations to the constitutive equations. Both of the two ways lead to the functionals

$$
I_{\mathbf{s}}[u] = \iint_{\mathscr{M}} \left[ \Sigma(\varepsilon(u)) - \Pi \right] dA - I_b[u_{\vert c}, \sigma(\varepsilon(u_{\vert c}))], \tag{5.7}
$$

$$
J_{8}[u] = -\iint_{\mathcal{M}} \left\{ \frac{1}{2} [\mathbf{T}(u, \sigma(\varepsilon(u))) \nabla + \mathbf{p}] \cdot \mathbf{u} + \frac{1}{2} \mathbf{Q}(u, \sigma(\varepsilon(u))) \nabla \cdot \mathbf{u} \right\}
$$
  
+  $(\Pi - \mathbf{p} \cdot \mathbf{u}) \right\} dA + \frac{1}{2} J_{b}[u_{|c}, \sigma(\varepsilon(u_{|c}))]$   
- $\frac{1}{2} \int_{\mathcal{G}_{f}} [(\Phi - \mathbf{B} \cdot \mathbf{u}) + \Psi] dS - \frac{1}{2} \int_{\mathcal{G}_{u}} (J \vec{v}^* \mathbf{\Gamma} \mathbf{M} \mathbf{v}) \vec{\mathbf{n}} \cdot \vec{v}^* dS,$  (5.8)

where the relations (5.6) and (4.5) have been employed additionally in the derivation of  $J_8$ . Their stationarity conditions are the static relations (ii) and (iii), expressed in displacements, and the geometric boundary conditions (v).

The solution to the problem

find 
$$
[u_0] \in \mathcal{U}
$$
 such that  $\delta I_8[u_0] = 0$  or  $\delta J_8[u_0] = 0$  (5.9)

is the actual displacement field of the deformed surface  $\overline{M}$ .

Finally, reversing the procedure underlying the derivation of the Hu-Washizu functional  $I_1$  from the initial functional I, we can exclude the geometric boundary conditions (v) from  $I_8$  to obtain the functional of the total potential energy

$$
\Pi[u] = \iint_{\mathcal{A}} \left[ \Sigma(\varepsilon(u)) - \Pi \right] dA - \int_{\varepsilon_f} (\Phi + \Psi) ds.
$$
 (5.10)

In fact,  $\Pi[u]$  could have been obtained directly from  $I[u, c]$  (3.13) via substitution of the kinematic relations  $\varepsilon = \varepsilon(u)$  there. Therefore the variation  $\delta \Pi$  coincides with the left-hand side of (2.10) after substitution of the relations  $\sigma = \sigma(\epsilon(u))$  for N and M. Consequently, the stationarity conditions of  $\Pi$  are the static relations (ii) and (iii) expressed in displacements. The associated variational problem can be stated as

find 
$$
[u_0] \in \mathcal{U}
$$
 such that  $\Pi[u_0]$  = stationary values of  $\Pi[u]$  (5.11)  
 $u_{\mu} \in \mathcal{D}_{\mu}$ 

where  $\mathcal{D}_u = {\mathbf{u}_{k} | g(u_{k}, u^*) = 0}.$  This is the stationary potential energy principle.

Closing this section let us remark that the elimination of the displacement field u expressed in the strains  $\varepsilon$  from the Hellinger-Reissner functionals  $I_2$  and  $J_2$  would lead to the complementary functionals and variational principles. However, the problem of inverting the nonlinear kinematic relations  $\varepsilon = \varepsilon(u)$  is nontrivial and has not yet been solved in the general case. The existing solution, due to Washizu[39], reaches only as far as the Donnel-Marguerre nonlinear theory of shallow shells. Clearly, the obstacle is of a geometric nature, stemming from the fact that more than one displacement state *u* may correspond to a given surface strain  $\gamma$  and from the coupling between the surface strain and change of curvature measures through the compatibility equations. It seems that the solution should be sought in modification of the attitude towards the deformation measures and the approach developed by de Veubeke provides with some sort of alternative here (see [21-23]).

## 6. FINAL REMARKS

The functionals and variational principles presented in the main body of this paper deserve some additional comments. First of all, they are grouped in two equivalent families exhibiting the properties of symmetry (cf. Fig. 1). The pairs  $I_k - J_k$  are related by the Stokes theorem. Their variations are always equal to each other:  $\delta I_k = \delta J_k$ . The functionals of the  $I$  family contain the geometric relations of the theory in the explicit form, whereas those of the  $\mathcal J$  family the static relations. Consequently, either of them lays down a suitable basis for the problems with some additional constraints of the geometric or static character, respectively (cf. [9]).

Secondly, the question concerning the character of the stationary points should be mentioned asstill an open one. So far the conditionsfor attainment ofinfima are thoroughly examined only in the theory of convex functionals[40] and the related theory of potential monotone operators[26, 32, 33]. Yet the variational problems arising in the general geometrically nonlinear theory of shells may turn out to be nonconvex due to the interference of the nonlinear strain-displacement relations or nonmonotonicity of the external loads. Hence, nonuniqueness and branching of solutions should be expected. The clarification of this question would be of essential importance for the problems of elastic stability.

Finally, if the external loads involved in the problem are nonpotential the incremental fonnulations can be employed. The variational principles presented herein should prove helpful even in this case. For these and related problems see [28].

*Acknowledgements-*The author wants to express his thanks to Professor H. Stumpf and his associates for their kind and helpful attitude during the author'sstay at the Ruhr-Universitit Bochum, where this article was written, and to Professor W. Pietraszkiewicz for his critical comments on the text.

#### 1174 M. L. SZWABOWICZ

#### REFERENCES

- l. K. Washizu, *Variational Methods in Elasticity and Plasticity,* 2nd edn. Pergamon Press, Oxford (1975).
- 2. N. P. Abovski, N. P. Andreev and A. P. Deruga, *Variational Principles of the Elasticity Theory and Shell Theory* (in Russian). Nauka, Moscow (1978).
- 3. L. A. Rozin, *Variational Formulation of Problems for Elastic Systems* (in Russian). Leningrad University Press (1978).
- 4. L. Y. Aynola, Varialional problems in the nonlinear theory of clastic shells (in Russian). *Prikl. Mal. Mekh.* 21, 339 (1957).
- 5. K. Washizu, Complementary variational principles in elasticity and plasticity. In *Duality andComplementarity in Mechanics ofSolids, Proc. Con! Duality in Problems ofthe Mechanics ofDeformed Bodies; Jablonna 1977.* Pol. Ac. Sci. Ossolineum. Wroclaw (1979).
- 6. H. Stumpf. The derivation of dual extremum and complementary stationary principles in geometrical nonlinear theory of shells. *Ing. Arch.* 48, 221 (1979).
- 7. H. Stumpf. Buckling and post-buckling ofshells for unrestricted and moderate rotations. In *Proc. Euromech-Colloquium* 165. Springer-Verlag. Berlin/New York (1984).
- 8. E. Stein, Variational functionals in the geometrical nonlinear theory and finite-element-discretizations with applications to stability problems. In *Theory of Shell,f, Proc. 3rd IUTAM Symp. on Shell Theory, 509, 1978.* North-Holland, Amsterdam (1980).
- 9. R. Schmidt and W. Pietraszkiewicz. Variational principles in the geometrically non-linear theory of shells undergoing moderate rotations. *Ing. Arch.* 50(3). 181 (1981).
- 10. K. Z. Galimov, *Foundations of the Nonlinear Thin Shell Theory* (in Russian). Kazan University Press (1975). 11. K. Z. Galimov. Some equations of the nonlinear shell theory (in Russian). In *Investigations in Plate and Shell Theory.* Kazan University Press (1981).
- 12. W. Pietraszkiewicz and M. L. Szwabowicz, Entirely Lagrangian nonlinear theory of thin shells. *Arch. Mech.* 33(2),273 (1981).
- 13. R. Schmidt. On variational formulations in the general geometrically non-linear first approximation theory of thin clastic shells. *ZAMM* 62, TI65 (1982).
- 14. R. Schmidt. Variational principles for general and restricted geometrically non-linear Kirchhoff-Love type shell theories. In *Proc. Int. Conf. on Finite Element Methods*, 621 (Edited by He Guangqian and Y. K. Cheung). Science Press. Beijing; Gordon & Breach. New York (1982).
- 15. R. Schmidt. Thin elastic shells undergoing small strains and large rotations-a simple consistent theory and variational principles. In *Proc. 2nd Int. Con! on Numerical Methodsfor Nonlinear Problems. Barcelona. 1984* (Edited by C. Taylor. E. Hinton and D. R. J. Owen). Vol. 2. pp. 17D-181. Swansea (1984).
- 16. R. Schmidt, On the geomctrically nonlincar analysis of thin clastic shells undergoing small strains and large rotations. In *SECTAM XII, the Southeastern Conf on Theoretical and Appl. Mech. Proc.•* Vol. 1, p. 564. Auburn University (1984).
- 17. M. L. Szwabowiez, On the potential boundary load in the nonlinear theory of thin shells. *Trans. Inst. Fluid FloII' Mach.* 86, 99 (1983).
- 18. W. Pietraszkiewiczand M. L. Szwabowicz, Hu-Wahizu functional for the Lagrangian geometrically nonlinear theory of thin elastic shells. *ZAMM* 62, Tl56 (1982).
- 19. M. L. Szwabowicz, The fundamental equations and the variational principles of the geometrically nonlinear thin shell theory (in Polish). In *Proc. of the IIIrd Polish Conf.*—Shell *Constructions—Theory and Applications*, Vol. 89, z. 18,267. Zeszyty Naukowe WSI w Opolu (1982).
- 20. M. L. Szwabowicz, Geometrically nonlinear thin elastic shell theory in the Lagrangian description (in Polish). Ph.D. thesis. Inst. of Fluid-Flow Mach., Gdańsk (1982).
- 21. J. G. Simmonds and D. A. Danielson, Nonlinear shell theory with finite rotation and stress-function vectors. *J. Appl. Mecl!. TrailS. ASME Sa. E34(4),* 1085 (1972).
- 22. S. N. Atluri. Alternate stress and conjugate strain measures. and mixed variational formulations involving rigid rotations, for computational analyses offinitely deformed solids. with application to plates and shells-I. Theory. *Comput. Structures* 18(1), 93 (1984).
- 23. R. Valid, Le principe des travaux virtuels et les principes variationnels associes. ONERA, Publication 1982- 1, Chatillon (1982).
- 24. R. Courant and D. Hilbert, *Methods of Mathematical Physics.* Vol. l. J. Wiley-Interscience. New York (1953).
- 25. C. Lanczos, *The Variational Principles ofMechanics.* 3rd edn. University of Toronto Press (1966).
- 26. J. T. Oden and J. N. Reddy, *Variational Methods in Theoretical Mechanics.* Springer-Verlag, Berlin (1976). 27. W. Pietraszkiewicz, Finite rotations in the nonlinear theory of thin shells. **In** *Thin Shell Theory: New Trends*
- *and Applications* (Edited by W. Olszak). CISM Courses and Lectures No. 240. Springer-Verlag. Wien (1980). 28. W. Pictraszkicwicz. Lagrangian description and incremental formulation in the non-linear theory of thin shells. *Int. J. Non-Linear Mech.* **19(2)**, 115 (1984).
- 29. W. T. Koiter. A consistent first approximation in the general theory of thin elastic shells. In *Theory of Thin Shells, Proc./UTAM Symp., Delft* 1959. North-Holland, Amsterdam (1960).
- 30. W. T. Koiter and J. G. Simmonds, Foundations ofshell theory. In *Theoretical and Appl. Mech.• Proc. 13th /UTAM Congr., Moscow* 1972. Springer-Verlag. Berlin (1973).
- 31. W. Pietraszkiewicz, *Finite Rotations and Lagrangean Description in the Non-Linear Theory of Shells.* Polish Scientific Publishers. Warszawa (1979).
- 32. M. M. Vainberg, *Variational Methods for the Study of Nonlinear Operators.* Holden-Day. San Francisco (1964).
- 33. M. M. Vainberg, *Variational Method and the Method of Monotone Operators in the Theory of Nonlinear Equations.* J. Wiley. New York (1973).
- 34. Guo Zhong-heng, Unified theory of variation principles in non-linear theory of elasticity. *Appl. Math. Mech.* 1, I (1980) (English Edition).
- 35. G. Romano, Potential operators and conservative systems. *Meccanica* 7.3 (1972).
- 36. W. T. Koiter, General equations of elastic stability for thin shells. In *Proc. Symp.: Theory of Shells, 187.* University of Houston (1967).
- 37. L.-P. Nolte and J. Makowski, Pressure loaded shells undergoing different levels of nonlinearity. *Mech. Res. Commun.* 12,6(1985).
- 38. J. G. Simmonds, Moment potentials. *Am.* J. *Phys.* **52,** 9 (1984).
- 39. K. Washizu, A note on the principle ofstationary complementary energy in nonlinear elasticity. In *Mechanics Today* (Edited by S. Nemat-Nasser). Vol. 5, p. 509. Pergamon Press. New York (1980).
- 40. I. Ekeland and R. Ternan. *Convex Analysis and Variational Problems.* North-Holland, Amsterdam; American Elsevier, New York (1976).